

E80 Spring 2016

Vibration and System Identification

(Adapted from previous years' lectures)

Prof. Angie Lee

E80 Lecture 10.1: Vibration and System ID

Remaining lectures

The Lecture Schedule							
Date	Tuesday	Date	Thursday				
19 JAN 2016	Flight Data Basics	21 JAN 2016	Data Fitting and Analysis				
26 JAN 2016	LabVIEW & MATLAB	28 JAN 2016	Basic Electrical Measurements				
2 FEB 2016	Op-amps & Signal Conditioning	4 FEB 2016	Temperature Measurements				
9 FEB 2016	Wind Tunnel & Fluid Measurements	11 FEB 2016	Thrust Measurements & Flight Modeling				
16 FEB 2016	Inertial Measurement	18 FEB 2016	Vibration & System ID				
23 FEB 2016	Atmospheric Science	25 FEB 2016	Sensors & Transducers				
8 MAR 2016	Flight Hardware						
5 APR 2016	Field Operations & Safety						

Start thinking about final project!

Folsam Dam: vibration testing

De Pietro Fellows 2004-2005



Nick von Gersdorff

Angie Cho







Eric Flynn



E80 Lecture 10.3: Vibration and System ID

Shaker: sinusoidal input



http://itll.colorado.edu/test_measurement_equipment/vibration_testing/

Cold gas thruster: impulse input



Lecture outline

- Interesting examples of rocket vibrations
- Intro to system identification and modal vibration
- Vibration analysis (math model, computational model)
- Vibration testing (experiment)
- Application: cantilever beam
- Application: rocket
- Fun video

- HEAT1X-Tycho Brahe inaugural flight
- Pilot's POV 9 Hz oscillation
- <u>http://www.youtube.com/watch?v=-</u> <u>rASHRBo9Rg&feature=player_embed</u> <u>ded</u>



Saturn rocket



"Pain was directly associated with motion of the eyeballs and testicles, as well as from internal heating that resulted from sloshing of the brain and viscera. The vibration frequency was also in the range of normal brain waves, adding confusion to decision making, hand and arm movement, and even speech."

Jim Fenwich on Pogo oscillations

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http://www.pwrengineering.com/articles/pogo.htm

Space shuttle main engine turbopumps

"The high-pressure pumps rotated at speeds reaching 36,000 rpm on the fuel side and 24,000 rpm on the oxidizer side. At these speeds, minor faults were exacerbated and could rapidly propagate to catastrophic engine failure." "...the vibration spectral data contained potential failure indicators in the form of discrete rotordynamic spectral signatures. These signatures were prime indicators of turbomachinery health..."

"Wings in Orbit" edited by Wayne Hale and Helen Lane

Rocket failure, March 2012

"While the lower stages of the North Korean rocket continued to function for several minutes, resonance at the top of the launch vehicle resulted in '**catastrophic disassembly** of the third stage at Max Q,' said Charles Vick, senior technical and space policy analyst at GlobalSecurity.org. 'The vibrations just tore it apart.'"



http://www.nytimes.com/2012/04/13/world/asia/north-korealaunches-rocket-defying-worldwarnings.html?pagewanted=all&_r=0

http://www.eetimes.com/electronics-news/4370955/Severe-vibrations-likely-brought-down-N--Korean-rocket

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Causes of rocket vibration

- Thrust oscillations
- Noise (pressure waves) due to motor or engine
- Fluid flow phenomena (aerodynamic stress)
 - Wind
 - Turbulence
 - Vortex shedding

Question

Why might we be interested in rocket vibrations?



E80 Lecture 10.10: Vibration and System ID

Intro to vibration analysis

• System identification:

building mathematical models of dynamical systems based on observed input-output data

• Modal analysis:

characterization of vibrational mode shapes and corresponding frequencies of a physical system

• Dynamic load and response:

load (input) applied dynamically (varying over time)

the response associated with this load is the <u>dynamic response</u> of the system

BASIC IDEA/GOAL

Determine the modal properties of a system: natural frequencies and modal shapes

Analysis of dynamic loading

How do you approach the problem of analyzing a structure with dynamic loading?

(a) Model your physical problem

Geometry, kinematics, material, loading

- (b) Derive governing equations (mostly differential equations)
- (c) Solve the equations
- (d) Interpret results and refine and repeat!

What else can you do to characterize a structure?

Take measurements!

Experimental studies to validate a model or help develop a model

A simple model: spring-mass-damper system

Around a resonance frequency, you can model as

$$m_{e} \ddot{y} = f - ky - c\dot{y}$$

$$m \ddot{y} + c\dot{y} + ky = f$$

$$\ddot{y} + \frac{c}{m_{e}} \dot{y} + \frac{k}{m_{e}} y = \frac{f}{m_{e}}$$

$$\ddot{y} + 2\zeta \omega_{n} \dot{y} + \omega_{n}^{2} y = f / m_{e}$$

$$\omega_{n} = \sqrt{\frac{k}{m_{e}}} \qquad \zeta = \frac{c}{2\sqrt{m_{e}}}$$



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Frequency response function (FRF)

Position

$$\frac{Y}{F} = \frac{\frac{1}{m_e} \left(\frac{1}{\omega_n}\right)^2}{1 - \left(\frac{\omega}{\omega_n}\right)^2 + 2\zeta \frac{\omega}{\omega_n} j}$$

Velocity

$$\frac{V}{F} = \frac{j\omega \frac{1}{m_e} \left(\frac{1}{\omega_n}\right)^2}{1 - \left(\frac{\omega}{\omega_n}\right)^2 + 2\zeta \frac{\omega}{\omega_n} j}$$

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Frequency response function (FRF)



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Damping coefficient

• From the peak
$$\omega_r = \omega_n \sqrt{1 - \zeta^2}$$

• From the half-power bandwidth $\Delta \omega = \omega_{+hp} - \omega_{-hp}$





http://www.sengpielaudio.com/calculator-cutoffFrequencies.htm

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Beams

- We can extend this idea from the spring-mass-damper example to a more complicated structural element, such as a beam
- Beams are one of the most important components in structural engineering
 - Examples of beams: bridges, walkways, rockets, ...



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Beams

Some important properties and characteristics of a beam are:

- 1) Cross sectional area: A, I
- 2) Length: L
- 3) Material: E, ρ



4) Supports (boundary conditions)

Туре	Boundary Conditions		
Pinned or hinged end	Left end: $w(0) = 0$, $-EI\frac{d^2w(0)}{dx^2} = 0$		
x = 0 $x = L$	Right end: $w(L) = 0$, $EI\frac{d^2w(L)}{dx^2} = 0$		
Clamped or fixed end	1 (0)		
	Left end: $w(0) = 0$, $\frac{dw(0)}{dx} = 0$		
x = 0 $x = L$	Right end: $w(L) = 0$, $\frac{dw(L)}{dx} = 0$		
Free end	Left end: $-EI \frac{d^2 w(0)}{dx^2} = 0, EI \frac{d^3 w(0)}{dx^3} = 0$		
x = 0 $x = L$	Right end: $EI \frac{d^2 w(L)}{dx^2} = 0, -EI \frac{d^3 w(L)}{dx^3} = 0$		
Sliding or guided end			
	Left end: $\frac{dw(0)}{dx} = 0$, $EI\frac{d^3w(0)}{dx^3} = 0$		
x = 0 $x = L$	Right end: $\frac{dw(L)}{dx} = 0$, $-EI\frac{d^3w(L)}{dx^3} = 0$		

https://www.ecomputingx.com/demo1.jsp

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Experiments: vibration testing

- Lab tests
 - Shaker tests

https://www.youtube.com/watch?v=o8H_NT7Ziao

• Impact hammer tests

https://www.youtube.com/watch?v=tBRjPN8m6zE

Cantilever beam (rotation lab)

• Mathematical model

to obtain natural frequencies and modal shapes

- Computational model
 example: SolidWorks model
- Experimental data

tap test in lab using strain gauges

Cantilever vibration modes



http://iitg.vlab.co.in/?sub=62&brch=175&sim=1080&cnt=1

Fig. 4.3: The first three undamped natural frequencies and mode shape of cantilever beam

https://www.youtube.com/watch?v=kun62B7VUg8

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Cantilever: computational model Mode 1: 299.09 Hz



E80 Lecture 10.22: Vibration and System ID

Mode 2: 1297.9 Hz



E80 Lecture 10.23: Vibration and System ID

Mode 3: 1417.6 Hz



E80 Lecture 10.24: Vibration and System ID

Mode 4: 1679.3 Hz



E80 Lecture 10.25: Vibration and System ID

Mode 5: 3917.6 Hz



E80 Lecture 10.26: Vibration and System ID

Mode 6: 5149.6 Hz



E80 Lecture 10.27: Vibration and System ID

Mode 7: 6538.1 Hz



E80 Lecture 10.28: Vibration and System ID

Mode 8: 7545.1 Hz



E80 Lecture 10.29: Vibration and System ID

Mode 9: 8377.9 Hz



E80 Lecture 10.30: Vibration and System ID

Mode 10: 8933.4 Hz



E80 Lecture 10.31: Vibration and System ID

Mode 11: 12199 Hz



E80 Lecture 10.32: Vibration and System ID

Mode 12: 13198 Hz



E80 Lecture 10.33: Vibration and System ID

Mode 13: 14941 Hz



E80 Lecture 10.34: Vibration and System ID

Mode 14: 17714 Hz



E80 Lecture 10.35: Vibration and System ID

Mode 15: 18072 Hz



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Cantilever: experiment

- Sensors: piezoelectric dynamic strain gauges
- Obtain data in time domain and in frequency domain
- Compare to analytical natural frequencies
- Build a low-pass filter to help analyze frequency data

Rocket (final project)

• Mathematical model

to obtain natural frequencies and modal shapes

- Computational model
 example: SolidWorks model
- Experimental data
 - tap test in <u>**lab</u>** using impact hammer and accelerometers</u>
 - sensors during *flight*

Rocket: mathematical model

general solution for a free-free beam is:



• calculations of a beam's cross sectional properties $A = \pi (r_o^2 - r_i^2)$

$$I = \frac{1}{4}\pi (r_o^{4} - r_i^{4})$$



Cross section of a hollow cylinder

"Structural Dynamics, The theory and applications," Joseph W. Tedesco, Addison Wesley, Longman Inc., 1999 E80 Lecture 10.39: Vibration and System ID

Rocket: mathematical model

• natural frequencies are:

$$\omega_n = \beta_n^2 \sqrt{\frac{EI_z}{\rho A}} = (\beta_n L)^2 \sqrt{\frac{EI_z}{\rho A L^4}}$$

Boundary Conditions	Frequency Equations	$\beta_1 L$	$\beta_2 L$	$\beta_2 L$			
Pinned-pinned	$\sin\beta L = 0$	3.141	6.282	9.423			
Fixed-free	$\cos\beta L \cosh\beta L + 1 = 0$	1.875	4.694	7.855			
Fixed-pinned (and pinned-free)	$\tan \beta L = \tanh \beta L$	3.927	7.069	10.210			
Fixed-fixed (and free-free)	$\cos\beta L \cosh\beta L = 1$	4.730	7.853	10.996			
Fixed-sliding (and free-sliding)	$\tan \beta L + \tanh \beta L = 1$	2.365	5.498	8.639			
Table 1. Natural Energy angles for Single Span Deems							

Table 1: Natural Frequencies for Single-Span Beams

Rocket: computational model Mode 1: 0 Hz



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Mode 2: 7.0439E-4 Hz



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Mode 3: 1.7816E-3 Hz



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Mode 4: 11.752 Hz



E80 Lecture 10.44: Vibration and System ID

Mode 5: 11.802 Hz



E80 Lecture 10.45: Vibration and System ID

Mode 6: 62.133 Hz



E80 Lecture 10.46: Vibration and System ID

Mode 7: 62.287 Hz



E80 Lecture 10.47: Vibration and System ID

Mode 8: 111.02 Hz



E80 Lecture 10.48: Vibration and System ID

Mode 9: 111.06 Hz



E80 Lecture 10.49: Vibration and System ID

Mode 10: 114.37 Hz



E80 Lecture 10.50: Vibration and System ID

Mode 11: 154.73 Hz



E80 Lecture 10.51: Vibration and System ID

Mode 12: 155.32 Hz



E80 Lecture 10.52: Vibration and System ID

Mode 13: 257.09 Hz



E80 Lecture 10.53: Vibration and System ID

Mode 14: 266.75 Hz



E80 Lecture 10.54: Vibration and System ID

Mode 15: 273.79 Hz



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Rocket: experimental data



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Rocket: experimental data

During *flight*: get time data from sensors

Post flight: analyze data in frequency domain compare to tap test results compare to model predictions (from analytical model and/or computational model)

Question

Why might we be interested in rocket vibrations?

- To characterize rocket's natural frequencies and modal shapes
- Avoid dead spots (nodes) to optimize sensor placement
- Design a vibration isolator to minimize vibrations in payload
- Validate your model





Video of flutter

https://www.youtube.com/watch?v=OhwLojNerMU